

Numerical Modeling of Coupled Thermo-elasticity with Relaxation Times in Rotating FGAPs Subjected to a Moving Heat Source

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Abstract. The time-stepping DRBEM modeling was proposed to study the 2D dynamic response of functionally graded anisotropic plate (FGAP) subjected to a moving heat source. The FGAP is assumed to be graded through the thickness. A Gaussian distribution of heat flux using a moving heat source with a conical shape is used for analyzing the temperature profiles. The main aim of this paper is to evaluate the difference between Green and Lindsay (G-L) and Lord and Shulman (L-S) theories of coupled thermo-elasticity in rotating FGAP subjected to a moving heat source. The accuracy of the proposed method was examined and confirmed by comparing the obtained results with those known previously.

Keywords: Thermo-elasticity; Functionally Graded Anisotropic Plates; Boundary Element Method.

1. Introduction

Biot [1] proposed the classical coupled thermo-elasticity (CCTE) theory to overcome the paradox inherent in the classical uncoupled thermo-elasticity (CUTE) theory that elastic changes have no effect on temperature. The heat equations for both theories are a diffusion type predicting infinite speeds of propagation for heat waves contrary to physical observations. A flux rate term into Fourier law of heat conduction is incorporated by Lord and Shulman (L-S) [2], who proposed an extended thermo-elasticity theory (ETE) which is also called as the generalized thermo-elasticity theory with one relaxation time. Another thermo-elasticity theory that admits the second sound effect is reported by Green and Lindsay (G-L) [3], who developed a temperature-rate-dependent thermo-elasticity theory (TRDTE) which is also called the generalized thermo-elasticity theory with two relaxation times by introducing two relaxation times that relate the stress and entropy to the temperature.

Functionally graded Plates (FGPs) are a type of non-homogeneous composites and the transient thermo-elastic problems for these non-homogeneous composites become important, and there are several studies concerned with these problems, such as Skouras et al. [4], Mojdehi et al. [5], Zhou et al. [6], Loghman et al. [7], Sun and Luo [8] and Mirzaei and Dehghan [9] which are papers involving functionally graded materials.

In recent years, the dynamical problem of thermo-elasticity for functionally graded anisotropic plates (FGAPs) becomes more important due to its many applications in modern aeronautics, astronautics, earthquake engineering, soil dynamics, mining

engineering, plasma physics, nuclear reactors and high-energy particle accelerators, for instance. Abd-Alla [10] obtained the relaxation effects on reflection of generalized magneto-thermo-elastic waves. Abd-Alla and Al-Dawy [11] obtained the relaxation effects on Rayleigh waves in generalized thermo-elastic media. Abbas and Abd-Alla [12, 13] studied generalized thermo-elastic problems for an infinite fibre-reinforced anisotropic plate. Xia, et al. [14] used a time domain finite element method to solve dynamic response of two-dimensional generalized thermo-elastic coupling problem subjected to a moving heat source based on Lord and Shulman theory with one thermal relaxation time

It is hard to find the analytical solution of a problem in a general case, therefore, an important number of engineering and mathematical papers devoted to the numerical solution have studied the overall behavior of such materials (see, e.g., El-Naggar et al. [15, 16], Abd-Alla et al. [17-19], Qin [20], Sladek et al. [21], Tian et al. [22], Fahmy [23-28], Fahmy and El-Shahat [29], Othman and Song, [30], Davi and Milazzo [31], Hou et al. [32], [Abreu et al. [33], Espinosa and Mediavilla, [34].

The advantages in the boundary element method (BEM) arises from the fact that the BEM can be regarded as boundary-based method that uses the boundary integral equation formulations where only the boundary of the domain of the partial differential equation (PDE) is required to be meshed. But in the domain-based methods such the finite element method (FEM), finite difference method (FDM) and element free method (EFM) that use ordinary differential equation (ODE) or PDE formulations, where the whole domain of the PDE requires discretization. Thus the dimension of the problem is effectively reduced by one, that is, surfaces for three-dimensional (3D) problems or curves for two-dimensional (2D) problems. And the equation governing the infinite domain is reduced to an equation over the finite boundary. Also, the BEM can be applied along with the other domain-based methods to verify the solutions to the problems that do not have available analytical solutions. Presence of domain integrals in the formulation of the BEM dramatically decreases the efficiency of this technique. Many different approaches have been developed to overcome these problems. It is our opinion that the most successful so far is the dual reciprocity boundary element method (DRBEM), which is the subject matter of this paper. The basic idea behind this approach is to employ a fundamental solution corresponding to a simpler equation and to treat the remaining terms, as well as other non-homogeneous terms in the original equation, through a procedure which involves a series expansion using global approximating functions and the application of reciprocity principles. However, there are some difficulties of extending the technique to several applications such as non-homogeneous, non-linear and time-dependent problems for examples. The main drawback in this case is the need to discretize the domain into a series of internal cells to deal with the terms not taken to the boundary by application of the fundamental solution. This additional discretization destroys some of the attraction of the method in terms of the data required to run the program and the complexity of the extra operations involved. The DRBEM is essentially a generalised way of constructing particular solutions that can be used to solve non-linear and time-dependent problems as well as to represent any internal source distribution. The DRBEM was initially developed by Nardini and Brebbia [35] in the context of two-dimensional dynamic elasticity and has been extended to deal with a variety of problems wherein the domain integral may account for linear-nonlinear static-dynamic effects. A more extensive historical review and applications of dual reciprocity boundary element method may be found in [Brebbia et al. [36], Wrobel and Brebbia [37], Partridge and Brebbia [38], Partridge and Wrobel [39] and Fahmy [40-47]].

97 The main objective of this paper is to study the model of two-dimensional equations of
 98 coupled thermo-elasticity with one and two relaxation times in rotating FGAPs subjected
 99 to a moving heat source. A predictor-corrector time integration algorithm was
 100 implemented for use with the DRBEM to obtain the solution for the temperature and
 101 displacement components. The accuracy of the proposed method was examined and
 102 confirmed by comparing the obtained results with the finite element method (FEM)
 103 results known before.

104 2. Governing equations of the FGAP

105 Consider a Cartesian coordinate system $Oxyz$ as shown in Fig. 1. We shall consider a
 106 rotating functionally graded anisotropic plate occupies the region $R = \{(x, y, z): 0 < x <$
 107 $y, 0 < y < \beta, 0 < z < \alpha\}$ with the boundary C and the material is functionally graded along the
 108 thickness direction. Thus, the governing equations of Coupled Thermo-elasticity with
 109 Relaxation Times can be written in the following form:

$$\sigma_{ab,b} - \rho(x+1)^m \omega^2 x_a = \rho(x+1)^m \ddot{u}_a, \quad (1)$$

$$\sigma_{ab} = (x+1)^m [C_{abfg} u_{f,g} - \beta_{ab}(T - T_0 + \tau_1 \dot{T})], \quad (2)$$

$$k_{ab} T_{,ab} = \beta_{ab} T_0 \dot{u}_{a,b} + \rho c(x+1)^m [\dot{T} + \tau_2 \ddot{T}] - Q. \quad (3)$$

110 where σ_{ab} is the mechanical stress tensor, u_k is the displacement, T is the temperature,
 111 C_{abfg} and β_{ab} are respectively, the constant elastic modulus and stress-temperature
 112 coefficients of the anisotropic medium, ω is the uniform angular velocity, k_{ab} are the
 113 thermal conductivity coefficients satisfying the symmetry relation $k_{ab} = k_{ba}$ and the
 114 strict inequality $(k_{12})^2 - k_{11}k_{22} < 0$ holds at all points in the medium, ρ is the density, c
 115 is the specific heat capacity, τ is the time, τ_1 and τ_2 are mechanical relaxation times, Q is
 116 the moving heat source.

117 3. Numerical implementation

118 Making use of (2), we can write (1) as follows

$$L_{gb} u_f = \rho \ddot{u}_a - (D_a T + \Lambda D_{a1f} u_f - \rho \omega^2 x_a) = f_{gb}, \quad (4)$$

119 The field equations can now be written in operator form as follows

$$L_{gb} u_f = f_{gb}, \quad (5)$$

$$L_{ab} T = f_{ab}, \quad (6)$$

120 Where the operators L_{gb} , f_{gb} , L_{ab} and f_{ab} are defined as follows

$$L_{gb} = D_{abf} \frac{\partial}{\partial x_b}, \quad f_{gb} = \rho \ddot{u}_a - (D_a T + \Lambda D_{a1f} u_f - \rho \omega^2 x_a) \quad (7)$$

$$D_{abf} = C_{abfg} \varepsilon, \varepsilon = \frac{\partial}{\partial x_g}, \Lambda = \frac{m}{x+1}, D_a = -\beta_{ab} \left(\frac{\partial}{\partial x_b} + \delta_{b1} \Lambda + \tau_1 \left(\frac{\partial}{\partial x_b} + \Lambda \right) \frac{\partial}{\partial \tau} \right)$$

$$L_{ab} = k_{ab} \frac{\partial}{\partial x_a} \frac{\partial}{\partial x_b}, \quad f_{ab} = \rho c(x+1)^m [\dot{T} + \tau_2 \ddot{T}] + \beta_{ab} T_0 \dot{u}_{a,b} - Q. \quad (8)$$

121 Using the weighted residual method (WRM), the differential equation (5) is transformed
 122 into an integral equation

$$\int_R (L_{gb} u_f - f_{gb}) u_{da}^* dR = 0. \quad (9)$$

123 Now, by choosing the fundamental solution u_{df}^* as the weighting function as follows

$$L_{gb}u_{df}^* = -\delta_{ad}\delta(x, \xi). \quad (10)$$

124 The corresponding traction field can be written as

$$t_{da}^* = C_{abfg}u_{df,g}^*n_b. \quad (11)$$

125 In which n_b is the unit normal vector to the surface.

126 The thermo-elastic traction vector can be written as follows

$$t_a = \frac{\bar{t}_a}{(x+1)^m} = (C_{abfg}u_{f,g} - \beta_{ab}(T - T_0 + \tau_1\dot{T}))n_b. \quad (12)$$

127 Applying integration by parts to (9) using the sifting property of the Dirac distribution,
128 and using equations (10) and (12), we can write the following elastic integral
129 representation formula

$$u_a(\xi) = \int_C (u_{da}^*t_a - t_{da}^*u_a + u_{da}^*\beta_{ab}Tn_b) dC - \int_R f_{gb}u_{da}^*dR. \quad (13)$$

130 The fundamental solution T^* of the thermal operator L_{ab} , defined by

$$L_{ab}T^* = -\delta(x, \xi). \quad (14)$$

131 By implementing the WRM and integration by parts, the differential equation (6) is
132 transformed into the thermal reciprocity equation

$$\int_R (L_{ab}TT^* - L_{ab}T^*T) dR = \int_C (q^*T - qT^*) dC, \quad (15)$$

133 Where the heat fluxes are as follows:

$$q = -k_{ab}T_{,b}n_a, \quad (16)$$

$$q^* = -k_{ab}T_{,b}^*n_a. \quad (17)$$

134 The thermal integral representation formula from (16) may be written as

$$T(\xi) = \int_C (q^*T - qT^*) dC - \int_R f_{ab}T^*dR. \quad (18)$$

135 The integral representation formulae of elastic and thermal fields (13) and (18) can be
136 combined to form a single equation as follows

$$\begin{aligned} \begin{bmatrix} u_d(\xi) \\ T(\xi) \end{bmatrix} = \int_C \left\{ - \begin{bmatrix} t_{da}^* & -u_{da}^*\beta_{ab}n_b \\ 0 & -q^* \end{bmatrix} \begin{bmatrix} u_a \\ T \end{bmatrix} + \begin{bmatrix} u_{da}^* & 0 \\ 0 & -T^* \end{bmatrix} \begin{bmatrix} t_a \\ q \end{bmatrix} \right\} dC \\ - \int_R \begin{bmatrix} u_{da}^* & 0 \\ 0 & -T^* \end{bmatrix} \begin{bmatrix} f_{gb} \\ -f_{ab} \end{bmatrix} dR. \end{aligned} \quad (19)$$

137 It is convenient to use the contracted notation to introduce generalized thermo elastic
138 vectors and tensors, which contain corresponding elastic and thermal variables as
139 follows:

$$U_A = \begin{cases} u_a & a = A = 1, 2, 3; \\ T & A = 4, \end{cases} \quad (20)$$

$$T_A = \begin{cases} t_a & a = A = 1, 2, 3; \\ q & A = 4, \end{cases} \quad (21)$$

$$U_{DA}^* = \begin{cases} u_{da}^* & d = D = 1, 2, 3; a = A = 1, 2, 3; \\ 0 & d = D = 1, 2, 3; A = 4; \\ 0 & D = 4; a = A = 1, 2, 3; \\ -T^* & D = 4; A = 4, \end{cases} \quad (22)$$

$$\tilde{T}_{DA}^* = \begin{cases} t_{da}^* & d = D = 1, 2, 3; a = A = 1, 2, 3; \\ -\tilde{u}_d^* & d = D = 1, 2, 3; A = 4; \\ 0 & D = 4; a = A = 1, 2, 3; \\ -q^* & D = 4; A = 4, \end{cases} \quad (23)$$

$$\tilde{u}_d^* = u_{da}^* \beta_{af} n_f. \quad (24)$$

140 The thermo-elastic representation formula (19) can be written in contracted notation as:

$$U_D(\xi) = \int_C (U_{DA}^* T_A - \tilde{T}_{DA} U_A) dC - \int_R U_{DA}^* S_A dR, \quad (25)$$

141 The vector S_A can be written in the split form as follows

$$S_A = S_A^0 + S_A^T + S_A^u + S_A^T + S_A^T + S_A^u + S_A^u, \quad (26)$$

142 Where

$$S_A^0 = \begin{cases} \rho \omega^2 x_a & a = A = 1, 2, 3; \\ Q & A = 4, \end{cases} \quad (27)$$

$$S_A^T = \omega_{AF} U_F \quad \text{with} \quad \omega_{AF} = \begin{cases} -D_a & A = 1, 2, 3; F = 4; \\ 0 & \text{otherwise}, \end{cases} \quad (28)$$

$$143 \quad S_A^u = -(D_{af} + \Lambda D_{a1f}) \mathcal{U} U_F$$

$$144 \quad \text{With} \quad \mathcal{U} = \begin{cases} 1 & a = A = 1, 2, 3; f = F = 1, 2, 3; \\ 0 & \text{otherwise}, \end{cases} \quad (29)$$

$$S_A^T = -\rho c(x+1)^m \delta_{AF} \dot{U}_F \quad \text{with} \quad \delta_{AF} = \begin{cases} 1 & A = 4; F = 4; \\ 0 & \text{otherwise}, \end{cases} \quad (30)$$

$$S_A^T = -\rho c(x+1)^m \tau_2 \delta_{AF} \ddot{U}_F, \quad (31)$$

$$S_A^u = -T_0 \mathbb{A} \delta_{1j} \beta_{fg} \varepsilon \dot{U}_F, \quad (32)$$

$$S_A^u = \dot{U}_F \quad \text{with} \quad = \begin{cases} \rho & A = 1, 2, 3; F = 1, 2, 3; \\ 0 & A = 4; f = F = 4. \end{cases} \quad (33)$$

145 The thermo-elastic representation formula (19) can be rewritten in matrix form as
146 follows:

$$[S_A] = \begin{bmatrix} \rho \omega^2 x_a \\ Q \end{bmatrix} + \begin{bmatrix} -D_a T \\ 0 \end{bmatrix} + \begin{bmatrix} -(D_{af} + \Lambda D_{a1f}) u_f \\ 0 \end{bmatrix} \\ + (\rho c(x+1)^m) \begin{bmatrix} 0 \\ T \end{bmatrix} - \rho c(x+1)^m \tau_2 \begin{bmatrix} 0 \\ T \end{bmatrix} - T_0 \begin{bmatrix} 0 \\ \beta_{ab} \dot{u}_{a,b} \end{bmatrix} + \begin{bmatrix} \rho \ddot{u}_a \\ 0 \end{bmatrix}. \quad (34)$$

147 By implementing the DRBEM to transform the domain integral in (25) to the boundary
148 integral, the source vector S_A in the domain was approximated by the following series of
149 given tensor functions f_{AN}^q and unknown coefficients α_N^q

$$S_A \approx \sum_{q=1}^N f_{AN}^q \alpha_N^q. \quad (35)$$

150 According to the implementation of the DRBEM, the surface of the plate has to be
151 discretized into boundary elements, where the total number of interpolation points is
152 $N = N_b + N_i$ in which N_b are collocation points on the boundary C and N_i are the interior
153 points of R

154 Thus, the thermo-elastic representation formula (25) can be written in the following form

$$U_D(\xi) = \int_C (U_{DA}^* T_A - \tilde{T}_{DA} U_A) dC - \sum_{q=1}^N \int_R U_{DA}^* f_{AN}^q dR \alpha_N^q. \quad (36)$$

155 By applying the WRM to the following inhomogeneous elastic and thermal equations:

$$L_{gb}u_{fn}^q = f_{an}^q, \quad (37)$$

$$L_{ab}T^q = f_{pj}^q, \quad (38)$$

Where the weighting functions were chosen to be the same as the elastic and thermal fundamental solutions u_{da}^* and T^* . Then the elastic and thermal representation formulae are as follows (Fahmy [42])

$$u_{de}^q(\xi) = \int_C (u_{da}^* t_{ae}^q - t_{da}^* u_{ae}^q) dC - \int_R u_{da}^* f_{ae}^q dR, \quad (39)$$

$$T^q(\xi) = \int_C (q^* T^q - q^q T^*) dC - \int_R f^q T^* dR. \quad (40)$$

The elastic and thermal representation formulae can be combined to form the following dual representation formulae

$$U_{DN}^q(\xi) = \int_C (U_{DA}^* T_{AN}^q - T_{DA}^* U_{AN}^q) dC - \int_R U_{DA}^* f_{AN}^q dR, \quad (41)$$

By substituting from (41) into (36), we can rewrite the dual reciprocity representation formula of coupled thermo elasticity as follows

$$U_D(\xi) = \int_C (U_{DA}^* T_A - \check{T}_{DA}^* U_A) dC + \sum_{q=1}^N \left(U_{DN}^q(\xi) + \int_C (T_{DA}^* U_{AN}^q - U_{DA}^* T_{AN}^q) dC \right) \alpha_N^q. \quad (42)$$

Using the thin plate splines (TPS) of Fahmy [27], we can write the particular solution of the displacement as follows

$$U_{GN}^q = \begin{cases} -\frac{4}{\lambda^4} \left[K_0(\lambda r) + \log(r) - \frac{r^2 \log r}{\lambda^2} - \frac{4}{\lambda^4} \right], & r > 0 \\ \frac{4}{\lambda^4} \left[\Upsilon + \log\left(\frac{\lambda}{2}\right) \right] - \frac{4}{\lambda^4}, & r = 0 \end{cases} \quad (43)$$

where K_0 is the Bessel function of the third kind of order zero, $\Upsilon = 0.5772156649015328$ is the Euler's constant and $r = \|x - \xi\|$ is the Euclidean distance between the field point x and the load point ξ .

According to the steps described in Fahmy [43], the dual reciprocity boundary integral equation (42) can be written in the following system of equations

$$\check{\zeta} \check{u} - \eta \check{t} = (\check{\zeta} \check{U} - \eta \check{\phi}) \alpha. \quad (44)$$

Where the matrix $\check{\zeta}$ contains the fundamental solution T_M^* and the matrix $\check{\zeta}$ contains the modified fundamental tensor $\check{T}_M^*$ with the coupling term.

The generalized displacements U_F and velocities $\dot{U}_F$ are approximated as follows [48]

$$U_F \approx \sum_{q=1}^N f_{FD}^q(x) \gamma_D^q, \quad (45)$$

$$\dot{U}_F \approx \sum_{q=1}^N f_{FD}^q(x) \tilde{\gamma}_D^q, \quad (46)$$

Where f_{FD}^q are tensor functions and γ_D^q and $\tilde{\gamma}_D^q$ are unknown coefficients.

The gradients of displacement and velocity were approximated as follows

$$U_{F,g} \approx \sum_{q=1}^N f_{K,g}^q(x) \gamma_K^q, \quad (47)$$

$$\dot{U}_{F,g} \approx \sum_{q=1}^N f_{FD,g}^q(x) \tilde{\gamma}_D^q. \quad (48)$$

176 If these approximations are substituted into equations (28) and (32) we obtain the
177 corresponding approximating source terms as follows

$$S_A^T = \sum_{q=1}^N S_{AD}^{T,q} \gamma_D^q, \quad (49)$$

$$S_A^{\dot{u}} = -T_0 \beta_{fg} \varepsilon \sum_{q=1}^N S_{AD}^{\dot{u},q} \tilde{\gamma}_D^q, \quad (50)$$

178 Where

$$S_{AD}^{T,q} = S_{AF} f_{FD,g}^q, \quad (51)$$

$$S_{AD}^{\dot{u},q} = S_{FA} f_{FD,g}^q. \quad (52)$$

179 Applying the point collocation procedure of Gaul, et al. [49] to equations (35), (45) and
180 (46) we have the following system of equations

$$\dot{S} = J\alpha, \quad U = J'\gamma, \quad \dot{U} = J'\tilde{\gamma}. \quad (53)$$

181 Similarly, the application of the point collocation procedure to the source terms equations
182 (29), (30), (31), (33), (49) and (50) leads to the following system of equations

$$\dot{S}^u = -(D_{af} + \Lambda D_{a1f}) \mathcal{U} U_F \quad \text{With} \quad (54)$$

$$\mathcal{U} = \begin{cases} 1 & a = A = 1, 2, 3; f = F = 1, 2, 3; \\ 0 & \text{otherwise,} \end{cases} \quad (54)$$

$$\dot{S}^T = \rho c(x+1)^m \delta_{AF} \dot{U}, \quad (55)$$

$$\dot{S}^{\ddot{T}} = -c\rho(x+1)^m \tau_2 \delta_{AF} \ddot{U}, \quad (56)$$

$$\dot{S}^{\dot{u}} = \tilde{A} \ddot{U}, \quad (57)$$

$$\dot{S}^T = \mathcal{B}^T \gamma, \quad (58)$$

$$\dot{S}^{\dot{u}} = -T_0 \beta_{fg} \varepsilon \mathcal{B}^{\dot{u}} \tilde{\gamma}. \quad (59)$$

185 Solving the system (53) for α , γ and $\tilde{\gamma}$ yields

$$\alpha = J^{-1} \dot{S}, \quad \gamma = J'^{-1} U, \quad \tilde{\gamma} = J'^{-1} \dot{U}, \quad (60)$$

186 Now, the coefficients α can be expressed in terms of nodal values of the unknown
187 displacements, velocities and accelerations as follows:

$$\alpha = J^{-1} (\dot{S}^0 + [\mathcal{B}^T J'^{-1} - (D_{af} + \Lambda D_{a1f}) \mathcal{U}] U + [\rho c(x+1)^m \delta_{AF} - T_0 \beta_{fg} \varepsilon \mathcal{B}^{\dot{u}} J'^{-1}] \dot{U} + [\tilde{A} - \rho c(x+1)^m \tau_2 \delta_{AF}] \ddot{U}), \quad (61)$$

188 Where $\tilde{A}$ and $\mathcal{B}^T$ are assembled using the sub matrices [] and ω_{AF} respectively.

189 Substituting from Eq. (61) into Eq. (44), we obtain

$$M \ddot{U} + \Gamma \dot{U} + K U = \mathcal{Q}, \quad (62)$$

190 In which $\ddot{U}, \dot{U}, U$ and $\mathcal{Q}$ represent the acceleration, velocity, displacement and external
191 force vectors, respectively, V, M, Γ and K represent the volume, mass, damping and
192 stiffness matrices, respectively, as follows:

$$\begin{aligned} V &= (\eta \tilde{\rho} - \zeta \tilde{\gamma}) J^{-1}, & M &= V [\tilde{A} - c\rho(x+1)^m \tau_2 \delta_{AF}], \\ \Gamma &= V [\rho c(x+1)^m \delta_{AF} - T_0 \beta_{fg} \varepsilon \mathcal{B}^{\dot{u}} J'^{-1}], \\ K &= \tilde{\zeta} + V [\mathcal{B}^T J'^{-1} + (D_{af} + \Lambda D_{a1f}) \mathcal{U}], & \mathcal{Q} &= \eta T + V \dot{S}^0, \end{aligned} \quad (63)$$

193 Using the following initial conditions

194 $U(0) = U_0, \dot{U}(0) = V_0.$

195 Then, from Eq. (62), we can calculate the initial acceleration vector W_0 as follows

$MW_0 = Q_0 - \Gamma V_0 - KU_0. \quad (64)$

196 An implicit-explicit time integration algorithm of Hughes et al. [50, 51], was developed
197 and implemented for use with the DRBEM. This algorithm consists in satisfying the
198 following equations

$M\ddot{U}_{n+1} + \Gamma^I \dot{U}_{n+1} + \Gamma^E \ddot{U}_{n+1} + K^I U_{n+1} + K^E \ddot{U}_{n+1} = Q_{n+1}, \quad (65)$

$U_{n+1} = \tilde{U}_{n+1} + \gamma \Delta \tau^2 \ddot{U}_{n+1}, \quad (66)$

$\dot{U}_{n+1} = \tilde{\dot{U}}_{n+1} + \alpha \Delta \tau \ddot{U}_{n+1}, \quad (67)$

199 Where the superscripts I and E denote, respectively, to the implicit and explicit parts and

$\tilde{U}_{n+1} = U_{n+1} + \Delta \tau \dot{U}_n + (1 - 2\gamma) \frac{\Delta \tau^2}{2} \ddot{U}_n, \quad (68)$

$\tilde{\dot{U}}_{n+1} = \dot{U}_n + (1 - \alpha) \Delta \tau \ddot{U}_n, \quad (69)$

200 Where we used the quantities $\tilde{U}_{n+1}$ and $\tilde{\dot{U}}_{n+1}$ to denote the predictor values, and U_{n+1}
201 and $\dot{U}_{n+1}$ to denote the corrector values. It is easy to recognize that the equations (66)-
202 (69) correspond to the Newmark formulas [52].

203 At each time-step, equations (65)-(69), constitute an algebraic problem in terms of the
204 unknown accelerations $\ddot{U}_{n+1}$

205 The first step in the code starts by forming and factoring the effective mass

$M^* = M + \gamma \Delta \tau C^I + \gamma \Delta \tau^2 K^I. \quad (70)$

206 The time step $\Delta \tau$ must be constant to run this step. As the time-step $\Delta \tau$ is changed, the
207 first step should be repeated at each new step. The second step is to form residual force

$Q_{n+1}^* = Q_{n+1} - C^I \tilde{U}_{n+1} - C^E \tilde{\dot{U}}_{n+1} - K^I \tilde{U}_{n+1} - K^E \tilde{\dot{U}}_{n+1}. \quad (71)$

208 The third step is to solve $M^* \ddot{U}_{n+1} = Q_{n+1}^*$ using a Crout elimination algorithm [53]
209 which fully exploits that structure in that zeroes outside the profile are neither stored nor
210 operated upon. The fourth step is to use predictor-corrector equations (66) and (67) to
211 obtain the corrector displacement and velocity vectors, respectively.

212 4. Numerical results and discussion

213 The Gaussian heat flux distribution $Q(x, y)$ can be expressed as

$Q(x, y) = \frac{3Q_0}{\pi r^2} e^{\left(-\frac{3(x^2+y^2)}{r^2}\right)} \quad (72)$

214 In which Q_0 is heat power of the plane heat source, r is the heat source radius.

215 Following Rasolofosaon and Zinszner [54] monoclinic North Sea sandstone reservoir
216 rock was chosen as an anisotropic material and physical data are as follows:

217 Elasticity tensor

218 $C_{abfg} = \begin{bmatrix} 17.77 & 3.78 & 3.76 & 0.24 & -0.28 & 0.03 \\ 3.78 & 19.45 & 4.13 & 0 & 0 & 1.13 \\ 3.76 & 4.13 & 21.79 & 0 & 0 & 0.38 \\ 0 & 0 & 0 & 8.30 & 0.66 & 0 \\ 0 & 0 & 0 & 0.66 & 7.62 & 0 \\ 0.03 & 1.13 & 0.38 & 0 & 0 & 7.77 \end{bmatrix} GPa \quad (73)$

219 Mechanical temperature coefficient

$$\beta_{ab} = \begin{bmatrix} 0.001 & 0.02 & 0 \\ 0.02 & 0.006 & 0 \\ 0 & 0 & 0.05 \end{bmatrix} \cdot 10^6 \text{ N / Km}^2 \quad (74)$$

220 Tensor of thermal conductivity is

$$k_{ab} = \begin{bmatrix} 1 & 0.1 & 0.2 \\ 0.1 & 1.1 & 0.15 \\ 0.2 & 0.15 & 0.9 \end{bmatrix} \text{ W / Km} \quad (75)$$

221 Mass density $\rho = 2216 \text{ kg/m}^3$ and heat capacity $c = 0.1 \text{ J/(kg K)}$. The numerical values
 222 of the temperature and displacement are obtained by discretizing the boundary into 120
 223 elements ($N_b = 120$) and choosing 60 well-spaced out collocation points ($N_i = 60$) in
 224 the interior of the solution domain, referring to the recent work of Fahmy [55, 56].

225 The initial and boundary conditions considered in the calculations are

$$226 \text{ at } \tau = 0, u_1 = u_2 = \dot{u}_1 = \dot{u}_2 = 0, T = 0 \quad (76)$$

$$227 \text{ at } x = 0 \quad \frac{\partial u_1}{\partial x} = \frac{\partial u_1}{\partial x} = 0, \frac{\partial T}{\partial x} = 0 \quad (77)$$

$$228 \text{ at } x = \underline{\gamma} \quad \frac{\partial u_1}{\partial x} = \frac{\partial u_1}{\partial x} = 0, \frac{\partial T}{\partial x} = 0 \quad (78)$$

$$229 \text{ at } y = 0 \quad \frac{\partial u_1}{\partial y} = \frac{\partial u_1}{\partial y} = 0, \frac{\partial T}{\partial y} = 0 \quad (79)$$

$$230 \text{ at } y = \underline{\beta} \quad \frac{\partial u_1}{\partial y} = \frac{\partial u_1}{\partial y} = 0, \frac{\partial T}{\partial y} = 0 \quad (80)$$

231 The present work should be applicable to any problems for coupled theory of thermo-
 232 elasticity in rotating FGAP. Such a technique was discussed in Fahmy et al. [57-60] who
 233 solved the special case from this study in the absence of a moving heat source. To
 234 achieve better efficiency than the technique described in Fahmy et al. [57-60], we use
 235 thin plate splines into a code, which is proposed in the current study. We extend the
 236 study of Fahmy et al. [57-60], to solve 2D in the presence of a moving heat source. Thus,
 237 it is perhaps not surprising that the numerical values obtained here are in excellent
 238 agreement with those obtained by Fahmy et al. [57-60]. The results are plotted in figures
 239 2-4 for the Green and Lindsay (G-L) theory and plotted in figures 5-7 for the Lord and
 240 Shulman (L-S) theory to show the variation of the temperature T and the displacement
 241 components u_1 and u_2 with x coordinate. We can conclude from these figures that the
 242 temperature T and the displacements u_1 decrease with increasing x but the displacements
 243 u_2 increase with increasing x for the two theories. It has been found that the comparison
 244 between these theories evaluates the effect of second thermal relaxation time taken by
 245 Green and Lindsay. These results obtained with the DRBEM have been compared
 246 graphically with those obtained using the finite element method (FEM) method of Xia et
 247 al. [14]. It can be seen from these figures that the DRBEM results are in excellent
 248 agreement with the results obtained by FEM, thus confirming the accuracy of the
 249 DRBEM.

250

251

252 5. Conclusion

253

254 A predictor-corrector implicit-explicit time integration algorithm was implemented for
 255 use with the DRBEM to obtain the solution for the temperature and displacement
 256 components of the two-dimensional problem of coupled thermo-elasticity theories with
 257 one and two relaxation times in rotating FGAP subjected to a moving heat source with a
 258 conical shape. The results obtained are presented graphically to show the difference

259 between Green and Lindsay (G-L) and Lord and Shulman (L-S) theories of coupled
 260 thermo-elasticity with relaxation times in rotating FGAP. The accuracy of the DRBEM
 261 results was examined and confirmed by comparing the obtained results with the FEM
 262 obtained results. It can be seen from these figures that the DRBEM results are in
 263 excellent agreement with the results obtained by FEM.

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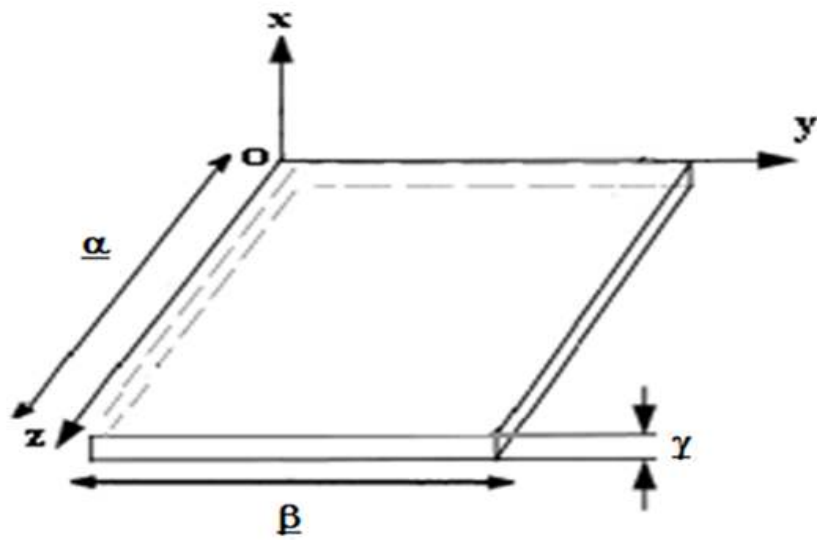


Fig. 1. The coordinate system of the FGAP.

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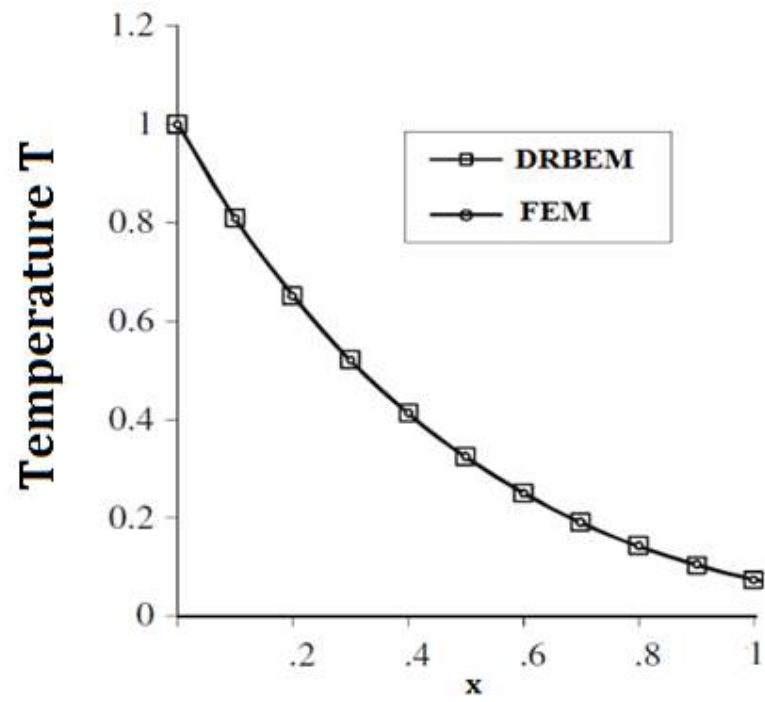


Fig. 2. Temperature distribution for G-L theory.

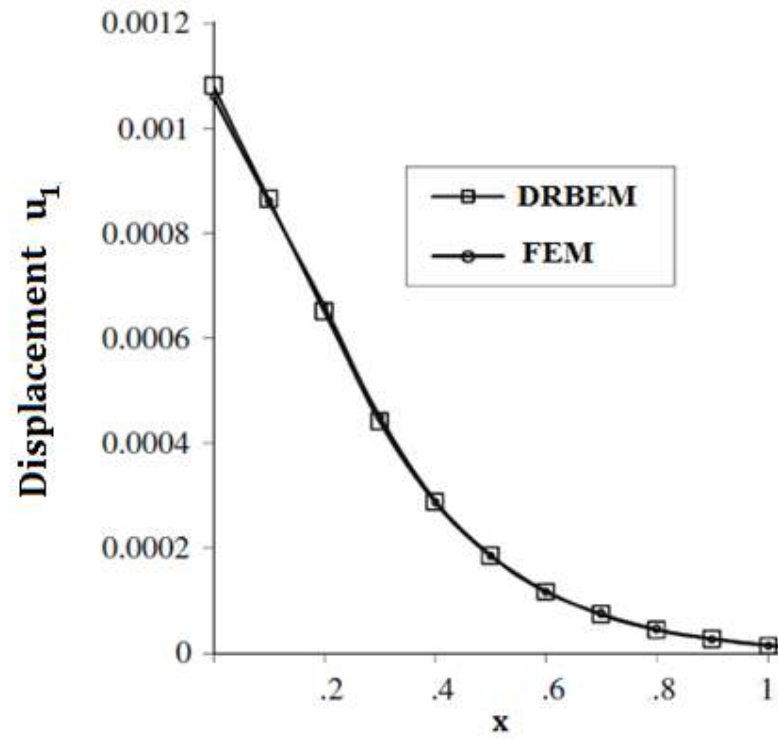


Fig. 3. Displacement distribution for G-L theory.

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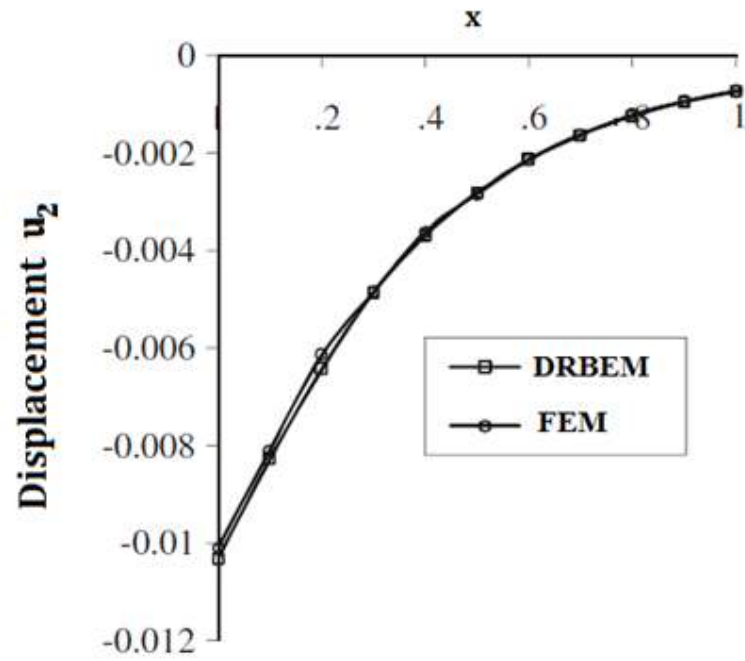


Fig. 4. Displacement distribution for G-L theory.

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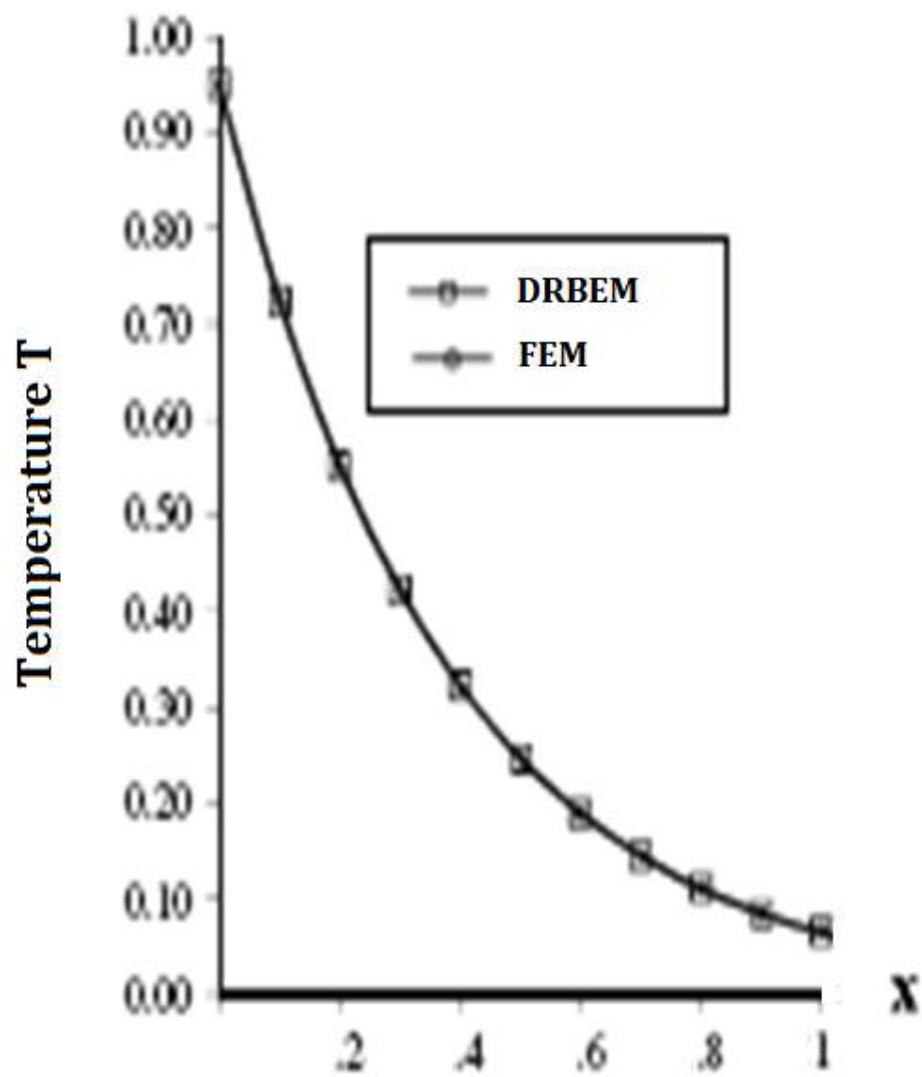


Fig. 5. Temperature distribution for L-S theory.

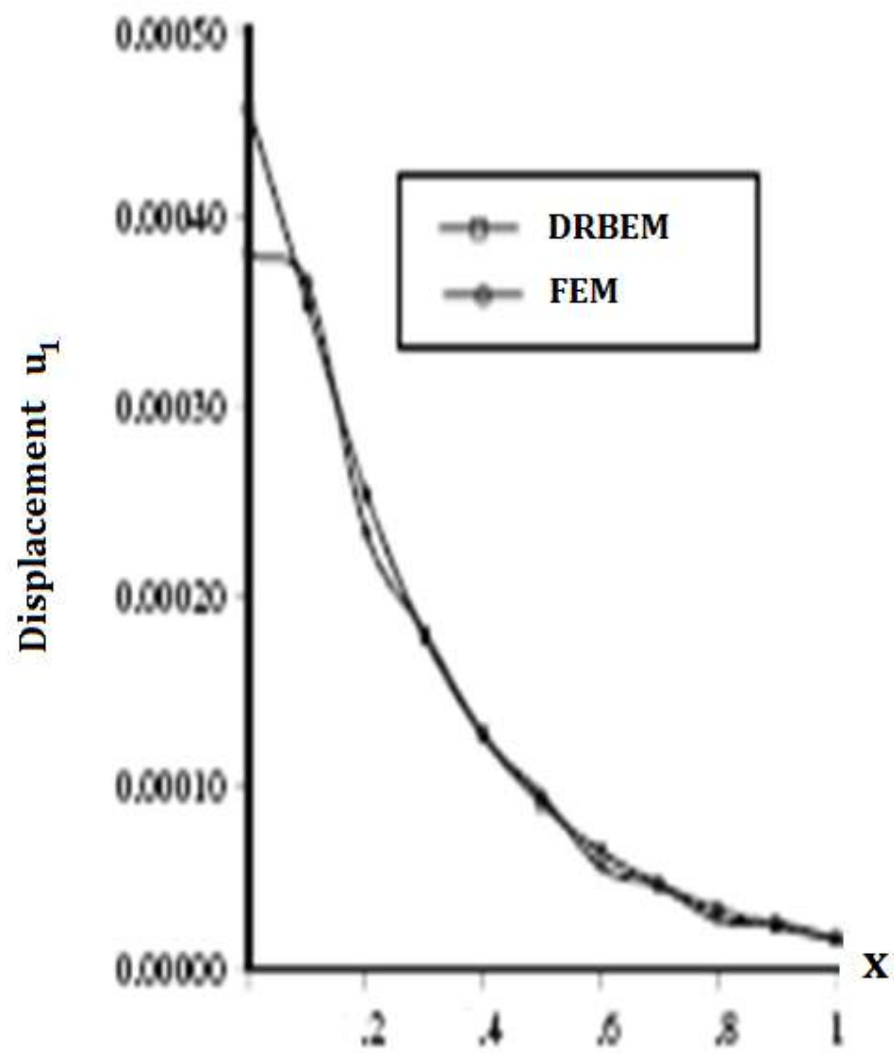


Fig. 6. Displacement distribution for L-S theory.

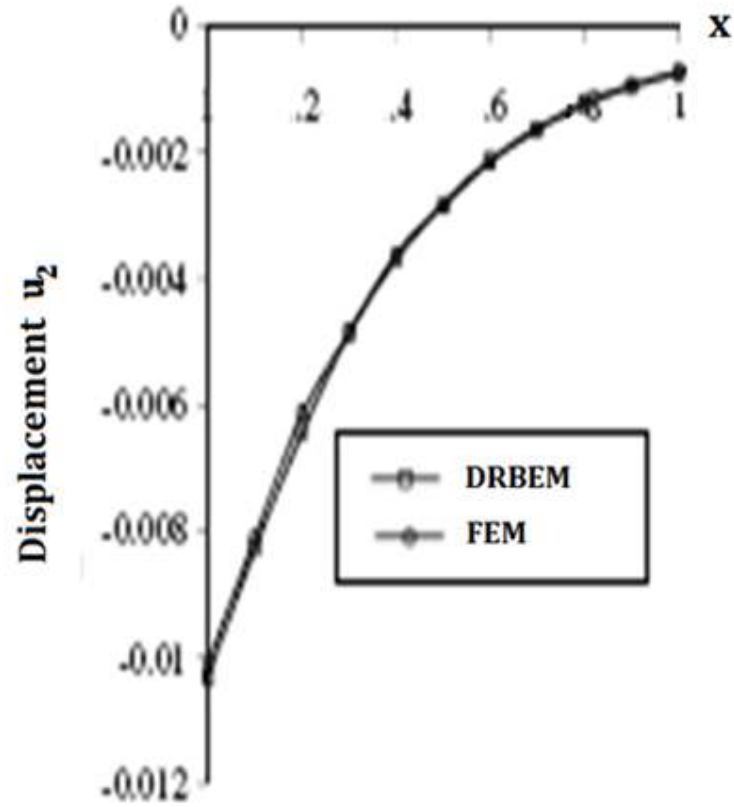


Fig. 7. Displacement distribution for L-S theory.

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